**Numerical Simulation of a Dual Pulse Solid Rocket Motor Flow Field**

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**ABSTRACT**

Numerical simulations are carried out for the internal flow field of a dual pulse solid rocket motor port to understand the flow behaviour. Three dimensional Reynolds Averaged Navier Stokes equations are solved alongwith shear stress transport turbulence model using commercial code. The combustion gas is assumed as a mixture of alumina and gases and single phase flow calculations are done with the thermo chemical properties provided for the mixture. The simulation captures all the essential features of the flow field. The flow accelerates through the pulse separation device (PSD) port and high temperature and high velocity gas is seen to impinge the motor wall near the PSD port. The overall total pressure drop through motor port and through PSD is found to be moderate.

**Keywords:** Numerical simulation, dual pulse solid rocket, pulse separation device

1. INTRODUCTION

For the past few decades, solid rocket motors have been extensively developed for tactical purposes and for civilian applications. Currently, different types of solid propellants rocket motors are used for different missions. Dual pulse rocket motor is one such variant of solid rocket where more than one pulse is used in the motor burn time. The first pulse of the rocket motor launches and accelerates the aerospace vehicle and after the burn out of first pulse, when the vehicle needs high manoeuvrability and power during its end game, the second pulse is fired to accelerate the missile for a second time. The better performance and higher manoeuvrability are achieved by using the same nozzle at the vehicle’s rear end for the two pulses. The aerodynamic performance of the missile is not affected by the operation of the second pulse motor.

The dual pulse solid rocket motor design consists of two burning chambers, separated by a bulkhead, designated as pulse separation device (PSD) and a nozzle. The pulse separation device protects the propellant grain in the second pulse chamber against high temperature and pressure impact during the first pulse operation. At initiation of the second pulse, the PSD reliably opens for the gas flow and the combustion gas from the second pulse chamber passes through the empty first pulse chamber and the nozzle. This design allows the initiation of the second pulse at any time after burn out of the first pulse. The use of one central nozzle for both pulses and the avoidance of lateral nozzles help the missile to show outstanding aerodynamic stability in manoeuvres during the second pulse phase. Number of dual pulse rocket motors were designed, manufactured and successfully tested in missile flights and the utility of this technology is demonstrated.

Schematic configuration of a dual pulse rocket motor is shown in Fig.1. The first pulse chamber is filled with a finocyl-shaped aluminized composite propellant. Its burn rate is moderate. In the second pulse chamber a star-shaped low aluminized composite propellant is cast. Its burn rate is high. Both chambers are screwed together. Between both chambers the PSD is jammed. A nozzle is attached to the rear of the first pulse chamber. Typical thrust time curve of a dual pulse rocket motor is shown in Fig. 2.

**Figure 1. Schematic of a dual pulse rocket motor.**

**Figure 2. Typical thrust-time history of a dual pulse rocket motor.**

Although, it is reported in the literature that the flow of reactive gases ejected from the regressing wall of a solid propellant rocket can be explained through Taylor–Culick equation, and Majdalani, et al. have adapted the theories for predicting the nozzle performances in both nozzle-
adapted and nozzleless configurations, the application of computational fluid dynamics (CFD) methods to simulate internal aerodynamics inside solid rocket motors (SRM) is advantageous. The numerical simulation of the internal flow in a motor usually involves a flow that ranges from the low subsonic in the head end of the motor to the supersonic regime of nozzle exit. Starting from internal three-dimensional inviscid flow in a Titan solid rocket motor by Johnston et al., much progress has been made in the CFD simulation of SRM which includes consideration of complicated geometry, viscous simulation, two phase calculation with solid particles, chemical reaction, combustion instability, etc.

In the present work, three dimensional viscous simulations are carried out in a dual pulse solid rocket motor port to understand the flow behaviour and to generate necessary thermo dynamical parameters for the design of rocket motor liner. The flow field is also analysed to estimate the pressure drop across the PSD.

2. COMPUTATIONAL METHODOLOGY

Three dimensional Reynolds Averaged Navier Stokes (RANS) equations are solved in a fully implicit manner using a commercial CFD software CFX-11 which is an integrated software system capable of solving diverse and complex multidimensional fluid flow problems. It is a finite volume method and is based on a finite element approach to represent the geometry. The method retains much of the geometric flexibility of finite element methods as well as the important conservation properties of the finite volume method.

The software has three major modules:

(a) Pre-processor - sets up the boundary condition and initial field condition,
(b) Solver manager - solves the flow field based on the grid and the boundary condition, and
(c) Post-processor - visualizes and extracts the results.

The software has a provision to use different numerical upwind schemes and it ensures global convergence of mass, momentum, energy and species. In the present study, the discretisation of the convective terms is done by second order upwind difference scheme. SST turbulence model is used for the turbulence closure. To find out the accuracy and the range of applications, the software has been validated extensively for various complex flows including supersonic base flows, transverse injection in supersonic flow for missile control, etc. and very good quantitative agreement with the experimental results is obtained. Recently, the same software was applied to predict the flow field of reacting gases coming out of a regressing wall of a solid rocket motor with composite propellant and the computed nozzle damping coefficients match with the experimental results for different throat to port area ratios. The details of governing equations, turbulence models and the discretisation schemes are given in the following subsections.

2.1 Governing Equations

The appropriate system of equations governing the turbulent compressible gas may be written as,

Continuity equation:

\[ \frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x_k} (\rho u_k) = 0 \quad k = 1, 2, 3 \]

Momentum equation:

\[ \frac{\partial}{\partial t} (\rho u_i) + \frac{\partial}{\partial x_k} (\rho u_i u_k) + \frac{\partial p}{\partial x_i} = - \frac{\partial}{\partial x_k} (\tau_{ik}) \quad , \quad i, k = 1, 2, 3 \]

Energy equation:

\[ \frac{\partial}{\partial t} (\rho E) + \frac{\partial}{\partial x_k} (\rho u_k H) = \frac{\partial}{\partial x_k} (\Gamma_j \frac{\partial T}{\partial x_j}) + \frac{\partial q_k}{\partial x_k} \]

where, \( \rho, u, p, E \), and \( H \) are the density, velocity components, pressure, total energy and total enthalpy respectively. Turbulent shear stress is defined as

\[ \tau_{ik} = \mu \left( \frac{\partial u_i}{\partial x_k} + \frac{\partial u_k}{\partial x_i} \right) \]

\( \mu = \mu_l + \mu_t \) is the total viscosity; \( \mu_l, \mu_t \) being the laminar and turbulent viscosity.

Laminar viscosity \( \mu_l \) is calculated from Sutherland law as

\[ \mu_l = \mu_{ref} \left( \frac{T}{T_{ref}} \right)^{3/2} \left( \frac{T_{ref} + S}{T + S} \right) \]

where, \( T \) is the temperature and \( \mu_{ref}, T_{ref} \) and \( S \) are known coefficients.

In eddy viscosity models, the stress tensor is expressed as a function of turbulent viscosity \( \mu_t \). Based on dimensional analysis, few variables \( k, \varepsilon, \omega \) are defined as given below,

Turbulent kinetic energy \( k \),

\[ k = \frac{u_i u_i}{2} \]

Turbulent dissipation rate \( \varepsilon \),

\[ \varepsilon = \frac{\partial u_i}{\partial x_j} \left( \frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) \]

Specific dissipation rate \( \omega \),

\[ \omega = \frac{\varepsilon}{k} \]

The turbulent viscosity \( \mu_t \) is calculated as

\[ \mu_t = C_\mu \frac{\rho k^2}{\varepsilon} \]

The heat flux \( q_k \) is calculated as \( q_k = - \lambda \frac{\partial T}{\partial x_k} \lambda \) is the thermal conductivity.

2.1.1 k-\omega Turbulence Model

The turbulent viscosity \( \mu_t \) is calculated as function of \( k \) and \( \omega \).

\[ \mu_t = f \left( \frac{\rho k}{\omega} \right) \]

Turbulent kinetic energy \( k \) equation:

\[ \frac{\partial}{\partial t} (\rho k) + \frac{\partial}{\partial x_k} (\rho k u_k) = \frac{\partial}{\partial x_k} \left( \Gamma_k \frac{\partial k}{\partial x_k} \right) + G_k - Y_k \]

Specific dissipation rate \( \omega \) equation:
\[
\frac{\partial}{\partial t}(\rho \omega) + \nabla \cdot (\rho \omega \mathbf{u}) = -\nabla \cdot \left( \Gamma_\omega \frac{\partial \omega}{\partial x_j} \right) + G_\omega - Y_\omega ABC
\]

where, \(G_\omega, Y_\omega, \Gamma_\omega\) and \(G_\omega, Y_\omega, \Gamma_\omega\) are the production, dissipation and diffusion terms of the \(k\) and \(\omega\) equations respectively.

2.1.2 Shear Stress Transport Turbulence Model

To retain the robust and accurate formulation of Wilcox’s \(k-\omega\) model in the near wall region, and to take advantage of the freestream independence of the \(k-\epsilon\) model in the outerpart of the boundary layer, Menter\(^{25}\) blended both the models through a switching function. \(k-\epsilon\) model was transformed into Wilcox’s \(k-\omega\) formulation and was multiplied by \((1-F_i)\) and added to original \(k-\omega\) model multiplied by \(F_i\). The blending function \(F_i\) will be one in the near wall region and zero away from the surface. In the second step, the definition of eddy viscosity was modified in the following way to account for the transport of the principal turbulent shear stress \((\tau = -\rho \omega \mathbf{u} \mathbf{v})\).

2.2 Discretisation of Governing Equations

The CFX-11 solver utilizes a finite volume approach, in which the conservation equations in differential form are integrated over a control volume described around a node, to obtain an integral equation. The pressure integral terms in momentum integral equation and the spatial derivative terms in the integral equations are evaluated using finite element approach. An element is described with eight neighbouring nodes. The advective term is evaluated using upwind differencing with physical advection correction. The set of discretised equations form a set of algebraic equations: \(A \mathbf{x} = \mathbf{b}\)

where \(\mathbf{x}\) is the solution vector. The solver uses an iterative procedure to update an approximated \(\mathbf{x}_n^*\) (solution of \(\mathbf{x}\) at \(n^{th}\) time level) by solving for an approximate correction \(\mathbf{x}^*\) from the equation \(A \mathbf{x}^* = \mathbf{R}\), where \(\mathbf{R} = \mathbf{b} - A \mathbf{x}_n^*\) is the residual at \(n^{th}\) time level. The equation \(A \mathbf{x}^* = \mathbf{R}\) is solved approximately using an approach called incomplete lower upper factorization method.

2.3 Computational Grid

Taking the advantage of the symmetry, only one-eighth sector the symmetry in geometry. The origin has been chosen at the centre of the head-end of pulse-2 motor. Two unstructured grids of 1.1 million and 1.4 million points have been made for the 1/8th geometry using ICEM-CFD\(^{23}\) grid generator to demonstrate the grid independence of the results (shown later). The grids are clustered near the critical sections of the geometry where large gradient of flow properties are expected like PSD region, near the steps provided at the entry of empty pulse-1 motor and nozzle throat regions. Hexahedral grids are used near solid boundary to capture boundary layer effects. Typical computational grid along with the computational boundary is shown in Fig. 3. It has two symmetry planes namely symmetry-1 (the plane passing middle of two adjacent PSD) and symmetry-2 (the plane passing the middle of PSD).

3. RESULTS AND DISCUSSIONS

CFX-11.0 software\(^{16}\) has been used for the numerical simulation of this problem. The inflow parameters used for the simulation are given in Table 1. Mass flow rate and total temperature inlet conditions are provided at the inlet (grain surface) of pulse-2 motor. The exit of the nozzle is kept as a supersonic outlet. There are two symmetry boundary conditions. Adiabatic and no slip conditions have been used on the walls. The exhaust gas constitutes solid/liquid alumina also due to combustion of aluminum particles. The thermo chemical properties are given for the mixture of the alumina and gases. The flow simulation is carried out assuming a single phase flow with the thermo chemical properties provided for the mixture. As mentioned earlier, shear stress transport (SST) turbulence model is used to model the turbulence. The simulations have been carried out till a convergence level of \(5.0 \times 10^{-5}\) for maximum residuals.

<table>
<thead>
<tr>
<th>Table 1. Inflow parameter for the simulation</th>
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<tr>
<td>Parameters</td>
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<td>---------------------------------------------</td>
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<tr>
<td>Total temperature (K)</td>
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<td>Ratio of specific heats</td>
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<tr>
<td>Thermal conductivity of the gas (W/m.K)</td>
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<tr>
<td>Molecular viscosity of the gas (Pa.s)</td>
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<td>Molecular weight of the gas</td>
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<tr>
<td>Specific heat at constant pressure for the gas (J/kg.K)</td>
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<tr>
<td>Total mass flow rate (kg/s)</td>
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</table>

The axial distribution of area average Mach number along the motor-PSD-nozzle assembly for two grids is presented in Fig. 4 to demonstrate the grid independence of the solution. The area average Mach number remains low throughout the motor but increases to supersonic value at the nozzle end. The increase in the Mach number at \(X/L = 0.36\) is due to the presence of PSD. A close agreement between the area averaged Mach numbers with two grids clearly indicates the grid independence of the solution. Static temperature at symmetry-2 plane along the length of the combustor is shown in Fig. 5. Mach number distributions at various axial planes in pulse-2 motor, PSD, pulse-1 motor and nozzle have been shown in Fig. 6.

Moderately high Mach number and velocity have been
found in the PSD port and adjacent core region of pulse-1 motor. Flow is found to expand at the convergent-divergent nozzle attached with pulse-1 motor due to which both Mach number and velocity are very high at the exit of the nozzle. To study the flow field around PSD, Mach number and velocity distributions at various axial planes in PSD and adjacent regions of pulse-1 motor have been presented in Fig. 7. The flow velocity is seen to increase through the PSD and reduces again due to increase in the area in 1st pulse motor. Velocity distribution on symmetry-1 and symmetry-2 planes depicted in Fig. 8 reveals that high velocity flow (~ 500 m/s) generated from the PSD port is hitting the walls of the various steps adjacent to the PSD before turning towards the centre of the pulse-1 motor. However, velocity on the inner wall of the pulse-1 motor adjacent to PSD regions is about 35 m/s - 45 m/s. The area-average axial velocity, static pressure and static temperature are evaluated at different axial locations and their axial distributions are presented in Fig. 9. Flow has been found to slightly expand in the PSD regions due to which slightly higher values of Mach number and velocity are observed in that region. However, static temperature has been found to be almost constant in pulse-2 motor, PSD and pulse-1 motor. Flow expanded towards the exit of the nozzle.
The details of the total pressure drop of the individual components are shown in the Fig. 10. It can be observed that the pressure drop across the pulse-2 motor along with the interface with PSD is about 4.11 per cent of the total pressure of pulse-2 motor \(P_{02m}\). The drops of total pressure in Pulse Separation Device (PSD) and its interface with step-1 are 2.18 per cent and 3.27 per cent, respectively. The drop in the 3-steps \(1^{st}, 2^{nd}, \text{and } 3^{rd}\) placed in between the PSD and pulse-1 motor is about 0.6 per cent. Whereas, the total pressure drop in the empty casing of the pulse-1 motor is 0.12 per cent. This low total pressure drop can be explained in the view of low velocities inside the pulse-1 motor casing. The total pressure drop inside the nozzle is about 6.66 per cent. The overall total pressure drop from the head end of pulse-2 motor to entry of the nozzle is found to be around 10.3 per cent.

The simulation captures all the essential features of the flow field. Grid independence of the simulation is demonstrated by comparing the flow variables with two different grids. The flow velocity is seen to expand through the PSD and reduces again due to increase in the area in 1\(^{st}\) pulse motor. High temperature combustion gas with velocity 500 m/s generated from the PSD port is seen to hit the walls of the various steps adjacent to the PSD. Flow velocity reduces significantly near the inner wall of the pulse-1 motor adjacent to PSD regions due to area increase. The computed flow field is analysed to obtain the pressure drop through various section of the motor geometry. It was found that out of total pressure drop of 10.3 per cent across the whole motor length, PSD and adjacent steps contribute about 6.05 per cent. The computed thermochemical variables in the motor chamber provide valuable input for the liner design of motor casing.

REFERENCES


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